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1 **Effect of trunnion roughness and length on the modular taper junction strength under**
2 **typical intraoperative assembly forces**

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25 off force

26 **Abstract**

27 Modular hip implants are at risk of fretting-induced postoperative complications most likely
28 initiated by micromotion between adjacent implant components. A stable fixation between
29 ball head and stem-neck taper is critical to avoid excessive interface motions. Therefore, the
30 aim of this study was to identify the effect of trunnion roughness and length on the modular
31 taper strength under typical intraoperative assembly forces.

32 Custom-made Titanium trunnions (standard/mini taper, smooth/grooved surface finish) were
33 assembled with modular Cobalt-chromium heads by impaction with peak forces ranging from
34 2 kN to 6 kN. After each assembly process these were disassembled with a materials testing
35 machine to detect the pull-off force as a measure for the taper strength.

36 As expected, the pull-off forces increased with rising peak assembly force ($p < 0.001$). For low
37 and moderate assembly forces, smooth standard tapers offered higher pull-off forces com-
38 pared to grooved tapers ($p < 0.038$). In the case of an assembly force of 2 kN, mini tapers
39 showed a higher taper strength than standard ones ($p = 0.037$).

40 The results of this study showed that smooth tapers provided a higher strength for taper junc-
41 tions. This higher taper strength may reduce the risk of fretting-related complications espe-
42 cially in the most common range of intraoperative assembly forces.

43

44 199 words (max. 200 words)

45

46 1. Introduction

47 Modular hip prostheses are commonly used in operation routines of total hip replacements
48 and offer at least one conical taper junction connecting the femoral stem-neck with the ball
49 head. This concept was established in the 1970s to allow surgeons more flexibility in the
50 choice of head material and diameters, and head-stem offsets for a more individualised ana-
51 tomical reconstruction of the patient's hip joint [1] while retaining the femoral stem and to
52 substantially reduce the inventory [2]. It was assumed that due to an optimized positioning of
53 the artificial joint the revision rates could be decreased. However, the latest clinical data do
54 not reflect the desired positive effects [3–6] with revision rates of up to 86 % for a double-
55 tapered modular hip prosthesis after a follow-up time of less than five years [4]. In recent
56 years concerns have arisen regarding fretting [7,8], wear [9,10] and corrosion at modular taper
57 junctions [8,9,11–15] further increasing the number of revision surgeries [5,8,9,12,16]. The
58 resulting postoperative complications include, but are not limited to, pain [7,13,17], soft tissue
59 damage [7,11], the formation of pseudotumours [7,15,18,19] and osteolysis [20,21] and are
60 frequently associated with high metal ion levels in the blood and/ or urine [15,22–24]. Despite
61 the fact that the precise failure mechanism at taper interfaces is not yet completely elucidated,
62 it is undisputed that micromotion between the adjacent implant components plays a role for
63 this clinical concern [9,25–28]. Previous experimental [28–33] and numerical studies [34,35]
64 have evaluated micromotion at taper interfaces: the documented values report a large range
65 from a few microns to more than 40 μm indicating that several factors such as, the prosthesis
66 geometry, manufacturing tolerances of the taper, the location of the taper connection (head-
67 stem or stem-neck), taper surface topography and the assembly conditions may influence the
68 micromotion levels [28–32,35]. These may be linked to changes in the location and size of the
69 taper contact area [36] and the assembly force. Taper junctions are exposed to high bending
70 and torsional loads during daily activities supporting the occurrence of micromotion. These

71 can provoke mechanical, as well as electrochemical initiated, processes in the fluid environ-
72 ment of the hip joint leading firstly, to fretting [27,37,38] and mechanically assisted crevice
73 corrosion [9,14,16,39] and secondly, to a cascade of adverse local tissue responses [5,13,40]
74 in the form of pseudotumours [18,19,39,15], allergic reactions and middle to high grade tissue
75 damage [11,15]. The material susceptibility to fretting and corrosion seems to be an important
76 factor in the failure mechanism as well. However, no consistent consensus currently exists
77 either for similar or for mixed material couplings [16,25,27,37,38]. Fretting-induced postop-
78 erative complications were first reported in significant numbers for large diameter metal-on-
79 metal hip joint articulations [11,41–43] and these device designs appear to negatively enhance
80 taper issues due to higher friction moments at the interface especially in case of low lubrica-
81 tion [44]. Besides fretting-induced complications, an insufficient taper strength caused, for
82 example, by an inadequate intraoperative assembly, may also provoke a loosening of the taper
83 connection [11]. Cases of disassembly of the ball head after dislocation [45,46] or during
84 closed reduction of a dislocated femoral component [47,48] have also been observed in clini-
85 cal applications.

86 The current state-of-the-art implies that a firm and permanent fixation of the implant compo-
87 nents is critical to minimize postoperative issues [10,16,25,49,50]. Although this fact is well
88 known, explicit guidelines to assemble the implant components are currently rarely available
89 in manufacturer's operative procedure guidelines [51–53]. Moreover, those handling instruc-
90 tions, for example, describing the procedure to assemble a ball head onto a stem taper, are
91 kept very vague [51–53]. It is hypothesised that design-, implantation- and surgeon specific
92 parameters may influence the risk of excessive interface motions and subsequently fretting
93 and corrosion due to inadequate assembly and fixation of these modular implants.

94 Therefore, the aim of this study was to determine the effect of taper surface roughness and
95 length on the stem-head taper junction strength under typical intraoperative assembly forces.

96 **2. Materials and methods**

97 *2.1. Materials, profilometry and assembly*

98 Three groups of titanium custom-made trunnions (Figure 1A, Ti6Al4V alloy, ASTM F136, in
99 total n = 15, Corin Group PLC, Cirencester, UK) with a 12/14 conical taper connection, dif-
100 ferent taper lengths and surface finishes were used for mechanical testing: smooth, standard
101 tapers (Group 1) vs. grooved, standard tapers (Group 2) vs. grooved, mini tapers (Group 3).
102 The taper length of the mini tapers was approximately 6.5 mm shorter compared to the stand-
103 ard tapers (14.5 mm) while retaining the taper size (maximum cone diameter 14 mm). Prior to
104 the assembly, the taper surfaces were cleaned with ethanol to remove any potential surface
105 contamination and the profile of the stem tapers' outer surface was scanned with a contact-
106 less, high-resolution, three-dimensional measurement instrument (ProScan2000 Surface Pro-
107 filometer, Scantron Industrial Products Ltd, Taunton, UK). Two different surface areas were
108 scanned per test sample with a scan area of 1 mm² and a step size of 0.002 mm in both direc-
109 tions each. Based on the scans, the average roughness values Rz and Ra were determined for
110 each trunnion. Additionally, the taper interface of the ball heads and trunnions were helically
111 scanned with a coordinate measuring machine using a ruby stylus for digitisation of the ge-
112 ometry (Incise, Renishaw, Gloucestershire, UK, Figure 1B and C). The surface profiling was
113 primarily used to determine the taper angles and to estimate the location of the press-fit and
114 the contact area (Figure 1B and C). The data sets were analysed using a custom script
115 (MATLAB R2011b; MathWorks, Natick, MA, USA). The centre of mass for each helix was
116 determined allowing the identification of the taper axis that was used as a basis for a subse-
117 quent best-fit algorithm. Due to a very robust algorithm, the proximal plane of the trunnions'
118 and the ball heads' plane at the open end, respectively, did not have to be aligned absolutely
119 horizontally during scanning, an angle deviation of up to 3° was acceptable. Based on the
120 outcome of this analysis, the taper angle difference, defined as the angle of the head subtract-

121 ed by the angle of the trunnion, was calculated (Figure 1B). The components were then as-
122 sembled at ambient environmental conditions with a 28 mm cobalt-chromium ball head (LC-
123 CoCr29Mo alloy, ASTM F1537, size L) by an impaction using a previously described cus-
124 tom-made drop-rig [54] to mimic the intraoperative procedure. The drop tower consisted of
125 two vertical sliders guiding a horizontal beam with a drop weight attached (total mass 2.4 kg).
126 The drop weight was capped with a nylon disc to reduce the risk of multiple impactions due
127 to a rebound effect. The drop rig was pre-calibrated in order to identify the relationship be-
128 tween drop height and peak assembly force. Based on the simulated peak assembly force the
129 drop height ranged between 22 mm and around 60 mm. Each trunnion-head pair was consecu-
130 tively assembled along the taper axis with different peak forces ranging from 2 kN to 6 kN
131 (sequence of assembly: $F_1 = 2$ kN, $F_2 = 2$ kN, $F_3 = 4$ kN, $F_4 = 2$ kN, $F_5 = 6$ kN, $F_6 = 2$ kN).
132 The assembly forces were chosen in alignment with typical intraoperative forces [55,56].

133

134 2.2. *Disassembly and statistics*

135 After each assembly process the implant components were disassembled using a materials
136 testing machine (Series 5965, Instron, Norwood, MA, USA) to measure the pull-off force as
137 an indicator for the taper strength (Figure 2). The trunnions were rigidly attached to the test-
138 ing machine base and almost the complete ball head was enclosed by a second fixture that
139 was directly coupled to the materials testing machine's actuator and the axial load cell (Figure
140 2). According to ISO 7206-10: 2003 the pull-off tests were performed at a stroke rate of
141 0.008 mm/s with a data acquisition rate of 10 Hz. In order to ensure that the pull-off forces
142 were not influenced by the consecutive test protocol, the results of all of the 2 kN tests were
143 statistically compared. The average pull-off force for each sample at a load level of 2 kN (F_1 ,
144 F_2 , F_4 , F_6) was calculated and then used for the following analyses to keep the sample size for
145 the assembly load levels constant ($n = 15$).

146 For statistical analyses non-parametric and parametric tests with a type-I-error probability of
147 $\alpha = 0.05$ were performed (SPSS Statistics 20, Munich, Germany). For further correlation
148 analyses, the pull-off forces were z-standardized leading to a variable's mean of zero and a
149 standard deviation of one. Correlations between two metric variables were assessed using
150 linear regression.

151

152 **3. Results**

153 *3.1. Profilometry & taper angles*

154 The trunnion surface showed a regular shaped pattern similar to a wave profile with a groove
155 depth of approximately 15.5 μm (grooved)/ 7.5 μm (smooth) and a groove spacing of around
156 300 μm (grooved) and 150 μm (smooth), respectively (Figure 3). Independent of the taper
157 length, grooved tapers had significantly higher roughness values in terms of both Rz
158 ($16.76 \pm 0.57 \mu\text{m}$ vs. $7.97 \pm 1.45 \mu\text{m}$, $p = 0.001$, Mann-Whitney U) and Ra ($4.14 \pm 0.54 \mu\text{m}$
159 vs. $2.92 \pm 0.44 \mu\text{m}$, $p = 0.003$, Mann-Whitney U) in z-axis than those with a smooth surface
160 finish (Figure 3). The ball heads exhibited on average a taper angle of $5.67 \pm 0.05^\circ$ leading to
161 a mean taper angle mismatch for standard trunnions of $0.10 \pm 0.05^\circ$. Since both, the standard
162 and the mini tapers, exhibited a 12/14 taper connection, the taper angles of the mini tapers
163 were larger compared to the standard tapers resulting in a taper angle mismatch of approxi-
164 mately zero ($-0.03 \pm 0.02^\circ$).

165

166 *3.2. Pull-off forces*

167 For all of the performed mechanical tests, the consecutive testing of the implant components
168 did not influence the pull-off forces at the trunnion-head interface ($0.216 \leq p \leq 0.922$, Krus-
169 kal-Wallis / one way ANOVA). Overall, independent of the taper length and the surface fin-
170 ish, the recorded pull-off forces were on average 24.9 % ($\pm 9.3\%$) of the assembly force. For

171 both surface modifications, the pull-off force of standard tapers increased significantly with
172 rising peak assembly force (Group 1 & 2, 2 kN: 0.549 ± 0.084 kN vs. 4 kN: 0.954 ± 0.101 kN
173 vs. 6 kN: 1.436 ± 0.145 kN, $p < 0.001$, two way ANOVA, Figure 4). For assembly forces of 4
174 kN or less, standard stem tapers with a smooth surface finish had significantly higher pull-off
175 forces compared to those with a grooved surface (Group 1 & 2, 0.820 ± 0.239 kN vs.
176 0.684 ± 0.202 kN, $p < 0.001$, two way ANOVA, Figure 4). Following a 6 kN impaction, no
177 influence of the surface finish on the pull-off force was detected (Group 1 & 2,
178 1.435 ± 0.145 kN, $p = 0.426$, one way ANOVA, Figure 4). Similar to the standard tapers, a
179 positive correlation between assembly and pull-off force was also determined for the mini
180 tapers (Group 3, linear regression, $\text{adj. } R^2 = 0.804$, $p < 0.001$). Only in case of an assembly
181 force of 2 kN, grooved mini tapers exhibited a significantly higher taper strength compared to
182 grooved standard tapers (Group 2 & 3, 0.630 ± 0.113 kN vs. 0.497 ± 0.041 kN, $p = 0.037$).
183 However, with rising assembly force this effect became smaller and smaller: at 4 kN still a
184 trend was observed (Group 2 & 3, $p = 0.065$, Welch Test) whereas for the highest assembly
185 force of 6 kN no effect could be found anymore (Group 2 & 3, $p = 0.266$, one way ANOVA,
186 Figure 4). Independent of the assembly force, for the standard tapers a significant influence of
187 the taper angle mismatch on the taper junction strength was not observed (Group 1 & 2,
188 $0.403 \leq p \leq 0.990$, linear regression). In contrast, the pull-off forces of the mini tapers tended
189 to increase with decreasing angular mismatch in the assessed range of -0.05° to -0.01° for all
190 assembly forces (Group 3, $0.022 \leq p \leq 0.093$, linear regression). Z-standardized pull-off forc-
191 es of mini tapers showed a significant negative correlation with the taper angle difference
192 (Group 3, $\text{adj. } R^2 = 0.718$; $p < 0.001$, Figure 5).

193

194

195 **4. Discussion**

196 Fretting-induced postoperative complications of modular hip prostheses have become a seri-
197 ous problem in total hip arthroplasty [57]. Micromotion between adjacent implant compo-
198 nents appears to be critical for this clinical concern [9,25–28]. In combination with fluid in-
199 gress into crevices resulting from angular differences of adjacent implant components or an
200 insufficient taper fixation, this may lead to fretting and/ or corrosion [9,16,27,37–39,15], dra-
201 matically limiting the functional life of a hip replacement [3–6,28]. Additionally, misalign-
202 ments and even a complete disassembly of a prosthesis component can be caused under ad-
203 verse circumstances by an insufficient taper strength [45–48]. Although these clinical prob-
204 lems are well documented, no explicit instructional guidelines to assemble the implant com-
205 ponents are provided for most of the implants available on the market. Since the importance
206 of a stable, rigid connection of taper junctions has been identified [10,16,25,49], this experi-
207 mental study focused on the impact of trunnion roughness and length on the taper junction's
208 strength under typical intraoperative assembly forces.

209 In the presented study only one taper design with a 12/14 taper was assessed with one specific
210 material coupling (Ti - CoCr). This fact may limit the transferability of the results to other
211 designs and/or material combinations. Due to differences in the taper geometry of cobalt-
212 chromium and ceramic ball heads, it is expected that the location of the press-fit, the size of
213 the contact area and the prevalent contact pressure will be different for the two materials. The
214 taper angle of ceramic ball heads is usually higher compared to metal heads suggesting that
215 the press-fit area is located nearer to the closed end of the taper connection. Therefore, a gen-
216 eral statement on the effect of the assessed influencing parameters cannot be easily drawn
217 without any further investigations. For the assembly process a custom-made drop tower was
218 used which utilised a plastic cap at the impactor's end. This scenario does not represent the
219 clinical situation in which the implant components were usually assembled by one or more

220 metal hammer blows. Due to the plastic end cap, the applied kinetic energy was reduced
221 compared to a metal-on-metal blow as a consequence of a damping effect. This study is fur-
222 thermore limited by assessing the surface topography of the trunnions only, as the profilome-
223 ter measurements of the head female tapers could not be made because of the nature of the
224 profilometer used in this study. The initial contact situation at the stem-head taper directly
225 after assembly was evaluated, rather than the assessment of changes in taper strength due to
226 any subsequent dynamic loading that may mimic the usual daily activity of a patient. Thus,
227 due to this lack of dynamic loading, potential interface micromotions were not recorded.

228 The surface topography of the tapers used in this experimental study was comparable to
229 threaded taper designs available on the market [17,58]; they offered a repetitive distinct sur-
230 face pattern with a specific groove height and spacing between two adjacent threads. The
231 rough tapers showed an average maximum profile height (R_z) comparable to the Profemur
232 (Wright Medical), Synergy (Smith and Nephew), Summit and Corail (DePuy Synthes) pros-
233 thesis with values between $16.02\ \mu\text{m}$ and $17.38\ \mu\text{m}$, however, their average roughness (R_a)
234 was lower ($4.14\ \mu\text{m}$ vs. $2.23 - 3.35\ \mu\text{m}$) [58]. The groove depth of the clinically used threaded
235 designs is smaller compared to the rough tapers ($7.24 - 13.49\ \mu\text{m}$ vs. $\approx 15.5\ \mu\text{m}$) [58]. The
236 roughness value R_a of the smooth tapers was comparable to the clinically used ones named
237 before as well as to the Trilock and Silent stem tapers (DePuy Synthes, $2.09 - 2.83\ \mu\text{m}$ vs.
238 2.92 ± 0.44), whereas, R_z conformed to the Secure-fit Max threaded design (Stryker,
239 $7.23\ \mu\text{m}$) and the non-threaded designs ABG II (Stryker), Taper-lock (Zimmer), Accolade
240 (Stryker) and SROM (DePuy Synthes, $6.1 - 7.5\ \mu\text{m}$) [58]. It should be noted, that the test
241 method used to determine the surface topography of the tapers deviated from the one applied
242 by Munir et al. [58]. In the present study only 2D characteristics were assessed, whereas Mu-
243 nir et al. used an interference microscope and a post-processing step to determine 3D topo-
244 graphical surface features as well.

245 This experimental study clearly demonstrated that the taper length and the surface roughness
246 can significantly influence the taper junction strength predominantly in the most common
247 range of intraoperative assembly forces. As expected and in agreement with other studies, the
248 pull-off forces increased significantly with rising peak assembly force [50,59–62]. A doubling
249 of the assembly force resulted in a 1.7 times higher pull-off force whereas a tripling gave rise
250 to a 2.6-fold increase of the taper strength. Rehmer et al., MacLeod et al. and Ihesiulor et al.
251 found a significant higher mean pull-off force/ assembly force ratio (25% vs. 33 - 67%) com-
252 pared to this study [50,61,63]: possible reasons may be differences in the topography and
253 roughness of the taper surfaces, the taper length and head size leading to discrepancies in the
254 location of the interlock and the contact force. Additionally, varying assembly procedures
255 (dynamic vs. static) and rigs, disassembly test speeds as well as different material combina-
256 tions may also have played a role [50,61,63]. The prosthesis design also affects the taper junc-
257 tion strength and its variability substantially [59]. The taper angle mismatch is suspected to be
258 of considerable importance as well [60]. A correlation between head size and taper strength
259 has already been found [61]: 36 mm metal heads exhibited a significantly lower pull-off force
260 compared to 28 mm ones when impacted with peak forces of 5 kN or less. This finding has
261 been associated with the high failures rates in large diameter hip replacements [61]. Previous
262 studies reported an increased seating of the head on the stem taper (primary seating) for high
263 assembly forces [33,62]. This seems to result in a more favourable taper contact situation with
264 a high contact pressure in the stem-head interfaces associated with the observed improved
265 taper strength [62] and a reduction in micromotion [33].

266 Following an impaction of 4 kN or less, smooth standard tapers exhibited a higher taper
267 strength, most probably due to a more favourable contact situation. The spacing between two
268 adjacent grooves and their depths is much smaller for the smooth surface finish compared to
269 the grooved tapers. Only in case of high assembly loads, small local plastic deformations of

270 the grooves are expected suggesting an increasing contact area comparable to smooth tapers.
271 Witt et al. found a significant increase in the area and the number of ridges being in contact
272 with rising assembly force [36]. In a similar manner, Fallahnezhad et al. determined in their
273 FE-Analysis an increasing contact length and pressure with rising assembly force, whereas
274 the contact length of CoCr/Ti couplings was always larger compared to CoCr/ CoCr junctions
275 [60]. Furthermore, a positive correlation between assembly force and the amount of perma-
276 nent plastic deformation has been reported [36]. Besides the reduced taper strength of rough
277 tapers following a slight or moderate impaction, these tapers seem to be also more susceptible
278 to fretting than smooth tapers [64]. Furthermore, a positive correlation between taper surface
279 roughness (R_{pk}) and wear rates has been reported [17]. In an in vitro study, Panagiotidou et
280 al. found a noticeable rupture of the oxide film during dynamic loading at a modular Ti/ CoCr
281 junction with a rough surface profile, whereas the fretting and corrosion damage for a smooth
282 taper was marginal [64].

283 Discrepancies in the taper strength between standard and mini tapers following a light ham-
284 mer blow may also be traced back to differences in the contact area and contact pressure. As
285 expected and in alignment with a FE-Analysis, the contact of standard tapers occurs proximal-
286 ly (closed end of taper connection) due to a positive angular mismatch [60,65], in contrast to
287 mini tapers with a mismatch of almost zero and therefore an unpredictable location of the area
288 being in contact (proximal, distal or across the whole taper length). This statement is in
289 agreement with a previously published study: Witt et al. found out that the location of the
290 damaged area is irregularly distributed along the whole taper surface in case of a small taper
291 angle difference [36]. Cook et al. expects an influence of the angular mismatch within the
292 stem-head taper connection on the generation of wear particle [10]. However, Kocagoz et al.
293 could not confirm a direct correlation between angular mismatch and wear and corrosion
294 scores [65]. But, the angular mismatch seems to be inversely correlated with the amount of

295 interface micromotions [33].

296 It is speculated that the intraoperative assembly procedure has a significant influence on the
297 initial contact situation [60] and subsequently a big influence for the clinical performance. In
298 addition to the assembly force, impaction angle, the number of impactions and the instrumen-
299 tation tool, the presence of contaminants in the interface due to an insufficient cleaning is also
300 considered as potentially critical. An inadequate assembly associated with the impact not
301 aligned axially may increase the presence of crevices within conical taper connections, allow-
302 ing fluid ingress and ultimately the creation of corrosion. During surgery some surgeons as-
303 semble the modular components by multiple impactions and under different conditions i.e.
304 wet or dry. It has been shown, that the impact force of the first hammer blow is the most im-
305 portant one with regard to the taper strength [50,59]. The assembly force sequence in case of
306 multiple impactions [59] and the taper condition prior to the assembly (wet or dry) may also
307 change the pull-off forces either in a positive or negative way, depending on the prosthesis
308 design [59]. The presence of bone chips contaminating the tapers can exhibit interface mi-
309 cromotions more than double that compared to clean tapers [30]. Weisse et al. demonstrated,
310 that contaminants such as bone chips, tissue or blood in the stem-head taper interface can re-
311 duce the static fracture load of ceramic ball heads by up to 90% compared to non-
312 contaminated interfaces [66]. It should furthermore be considered that, depending on the sur-
313 gical approach chosen, a dynamic assembly of the stem-head taper junction with a hammer or
314 a head impactor can be excessively difficult or impossible. It can also be speculated that the
315 tapers cannot be easily cleaned to remove any contaminants in the interface prior to the as-
316 sembly [57]. To the authors' knowledge there are currently no data available implying that the
317 number of fretting complications directly correlates to a specific surgical approach. This sug-
318 gests that not only the assembly force and condition (wet or dry) but rather several factors
319 affecting the risk of fretting e.g. the taper design, angular mismatch, head diameter and the

320 used materials. Nevertheless, for further developments of modular components this issue
321 needs to be adequately addressed.

322 Although there are currently no data available confirming that a high taper strength is directly
323 linked with a reduced risk of fretting complications, it seems to be highly probable that this
324 hypothesis is true. Fretting can only occur if there are crevices present, which allow fluid in-
325 gress [27]. In case of a high taper strength accompanied with an extremely high contact pres-
326 sure the crevices within the taper junction may be negligibly small avoiding fluid ingress and
327 preventing the initiation of corrosion. A few experimental in vitro studies have already as-
328 sessed the influence of assembly load, axial load and assembly condition (wet vs. dry), re-
329 spectively, on the onset of fretting. Goldberg et al. and Mroczkowski et al. assessed the fret-
330 ting corrosion behaviour using an in vitro electrochemical test set-up [27,49]. The open circuit
331 potential (OCP) decreased with rising cyclic load whereas the fretting current increased. This
332 can be seen as a result of a removal of the oxide film and a subsequent repassivation within
333 the taper connection [49]. The force threshold to initiate taper fretting is significantly higher
334 for implants assembled with a strong impaction (6.7 - 8.0 kN) in air (onset at a load of ≈ 2.5
335 kN [49]) than those pressed only by hand (for wet and dry condition, onset at a load less than
336 0.5 kN [49]) or statically assembled with 2.0 kN (onset at a load less than 1.3 kN [27]). How-
337 ever, during daily living activities, modular taper connections are not only exposed to pure
338 axial loads but rather to a combination of axial and rotational loads. In addition, a positive
339 correlation between assembly force and the minimum torque required to initiate fretting pro-
340 cesses in the interface has already been reported [54]. The determined values for load and
341 torque at the onset of fretting can easily be reached during daily living activities [27,54].
342 Baxmann et al. showed with an in-vitro fretting test system that fretting wear with indications
343 of particle detachment can occur in case of a low contact pressure (normal load $\leq 50\text{N}$) com-
344 bined with high interface motions ($\geq 25\text{N}$) [67]. Chu et al. found out in their FE Analysis that

345 a separation of contact in the taper connection can cause wear and corrosion [68]. Based on
346 the current knowledge, a correlation between taper strength and fretting damage seems possi-
347 ble but cannot be directly deduced.

348 As already mentioned by MacLeod et al., the initial contact situation directly after the assem-
349 bly procedure does not permit direct conclusions on the long-term performance [61]. Never-
350 theless, the taper strength may be one of several contributors to estimate the risk of fretting.

351

352 **5. Conclusions**

353 This study has demonstrated that trunnion-specific parameters as well as the assembly force
354 have a significant impact on the stem-head taper strength. High assembly forces gave rise to a
355 greater pull-off force; this may decrease the interface micromotions and ultimately the risk of
356 fretting and wear. Nevertheless, it should be considered that an excessively high hammer
357 blow may provoke damage to the bony structure and/or the surrounding tissue during surgical
358 assembly. Therefore, an assembly force of around 4 kN appears to be a reasonable compro-
359 mise in agreement with the previous recommendation made by Rehmer et al. [50] and
360 Haschke et al. [33].

361 An important finding is that smooth tapers are more appropriate to use in taper connections
362 with modular metal heads in the future since these tapers offer a higher taper strength, espe-
363 cially, in the most common range of intraoperative assembly forces. It is furthermore suspect-
364 ed, that rough tapers are more susceptible to fretting than smooth ones [64]. The results also
365 indicate that mini tapers can exhibit comparable taper strength to those of standard tapers
366 even if they offer an overall smaller taper surface area. There is thus evidence that it is not the
367 overall taper surface area that is essential, but rather the actual taper contact area, which is
368 affected by, but not limited to the surface topography, taper angle mismatch and the assembly
369 force.

370 **Ethical approval**

371 Not required.

372

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375

376 **Conflict of interest statement**

377 All authors do not have any conflicts of interest that are related to this study, to disclose.

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